

e-ISSN: 2319-8753 | p-ISSN: 2347-6710

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International Journal of Innovative Research in SCIENCE | ENGINEERING | TECHNOLOGY

# INTERNATIONAL JOURNAL OF INNOVATIVE RESEARCH

IN SCIENCE | ENGINEERING | TECHNOLOGY

Volume 13, Issue 5, May 2024

INTERNATIONAL STANDARD SERIAL NUMBER INDIA

### Impact Factor: 8.423

9940 572 462

6381 907 438

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|e-ISSN: 2319-8753, p-ISSN: 2347-6710| <u>www.ijirset.com</u> | Impact Factor: 8.423| A Monthly Peer Reviewed & Refereed Journal |

|| Volume 13, Issue 5, May 2024 ||

DOI:10.15680/IJIRSET.2024.1305028

## **Reduce the Frequency Amplitude and Road Condition Implement on Vehicle Chassis**

Yeduru Jaya Santhoshi Kumari<sup>1</sup>, Piradi Dhanaraju<sup>2</sup>, Nandipalli Venkata Sai Vivek<sup>3</sup>,

Nambala Sitharam<sup>4</sup>, Nadampalli Venkata Narsimha Raju<sup>5</sup>, Malla Pradeep<sup>6</sup>

Assistant Professor, Department of Mechanical Engineering, Avanthi Institute of Engineering and Technology,

Tamaram, Narsipatnam, India<sup>1</sup>

U.G. Student, Department of Mechanical Engineering, Avanthi Institute of Engineering and Technology, Tamaram,

Narsipatnam, India<sup>23456</sup>

**ABSTRACT**: The chassis stands as a crucial and integral component within the framework of a vehicle. The primary objective of this undertaking revolves around conducting static structural analysis as well as harmonic analysis to explore the fundamental characteristics of a truck chassis. In the realm of stationary response, the focus is on identifying the location of failure zones and areas with high stress concentrations. Simultaneously, this analysis aims to determine the extent of deformation experienced by the truck chassis.For a comprehensive understanding of the dynamic response of the truck chassis, factors such as natural frequencies and mode shapes are meticulously planned through the utilization of the Finite Element Analysis (FEA) technique. The ensuing numerical analysis is then devised to validate the results obtained from the FEA models.To enhance the design and performance of the chassis, strategic modifications are envisaged. This involves the incorporation of a tuned mass damper to optimize the vibrations associated with the truck chassis and, in turn, mitigate the likelihood of failure. The ultimate objective is to curtail the frequency amplitude, particularly under favorable road conditions. Through these interventions, the goal is to achieve a chassis design that not only meets structural integrity requirements but also minimizes vibrations and enhances overall performance on the road.

**KEYWORDS**: **Truck** Chassis,Lader Frame,Stress, Strain, Deformation,Finite Element Analysis, CATIA,ANSYS. Structural Strength design, Structural interaction with fluid runs, Analysis of shock (underwater & in materials), Mass diffusion, Buckling problems, Dynamic analyses, Electromagnetic evaluations, Metal forming, Coupled analyses

#### I. INTRODUCTION

The truck chassis serves as a foundational structure for business vehicles, enduring significant loads and forces. Dynamic analysis is crucial to prevent resonance and structural failure during operation, minimizing vibration and noise issues. The chassis typically consists of longitudinal and cross members, with an upswept frame to accommodate axle movement and maintain low height. The front overhang extends beyond the front axle, while the rear overhang extends past the rear axle. Commercial vehicles often employ framed construction for higher ground clearance and steering flexibility. The engine, clutch, and transmission form a rigid assembly mounted at the front end of the frame, isolated from shocks and vibrations by rubber blocks. Various cross-sections, such as channel and square box sections, provide bending stiffness for side members, often joined by riveting or lap welding for heavy-duty applications.

#### **II. RELATED WORK**

M. Ravi Chandra etal says here existing TATA 2515EX vehicle is taken for displaying and examination of an overwhelming vehicle body. The three different composite substantial vehicle skeleton structure have been displayed for three diverse cross-areas. For example, C, I and Box type cross areas And likewise with regards to different composite materials named as Carbon/Epoxy, Eglass/Epoxy and S-glass/Epoxy. After experienced examination with the assistance of ANSYS 12.0 they found that whole outcomes said as polymeric composite substantial vehicle undercarriage of I area for a similar burden conveying limit a decrease in weight of 73%~80%, common recurrence of PCHVC overwhelming vehicle case are 32%~54% higher than substantial vehicle steel body and 66~78% stiffer than the substantial vehicle steel skeleton.

International Journal of Innovative Research in Science, Engineering and Technology (IJIRSET)



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#### **III. METHODOLOGY**

The finite element method {is a|is just a|is really a} numerical technique for solving engineering problems. It {is a|is just a|is really a} powerful analysis tool used {to solve|to resolve|to fix} simple {as well as|in addition to|along with} complicated problem. 1. Create 3D CAD model: The 3D modelling of the chassis is done|is performed|is completed} using CATIA V5 software and saved in a IGES format. 2. Importing: Import the CAD geometry to the FEA package ANSYS 19.0 3. Material properties: The material properties are defined in the FEA package.

|               |       | AISI | ASTM |          |
|---------------|-------|------|------|----------|
|               | UNITS | 4130 | A710 | AISI 317 |
| Е             | G.Pa  | 205  | 210  | 193      |
| Poission raio |       | 0.29 | 0.29 | 0.27     |
| Density       | Kg/m3 | 7850 | 7850 | 7890     |

Meshing: In this operation the CAD geometry is divided into large {number of|quantity of|amount of} small pieces called mesh. The auto generate mesh feature of ANSYS 19.0 is used in this research due to the limited resources at the disposal of the researcher. Defining boundary conditions: The loads are applied and the position of the loadis defined. The constraints and the supports provided are used as input. Solve: The FEA package ANSYS 19.0 solves the model with the given mesh and loads for static, dynamic and harmonic analysis. Post processing: The reviewing of the results and the solutions are carried out in ANSYS 19.0 Itself. The results are|answers are viewed in various formats such as graph, values and animations.

#### CATIA MODEL

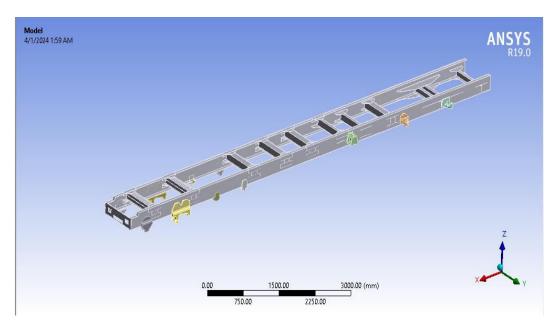


Fig1: Ladder Chassis

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#### **IV. EXPERIMENTAL RESULT**

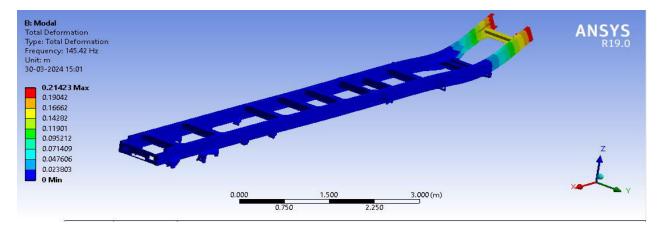


Fig 2 : mode 1 natural frequency 145.4 Hz

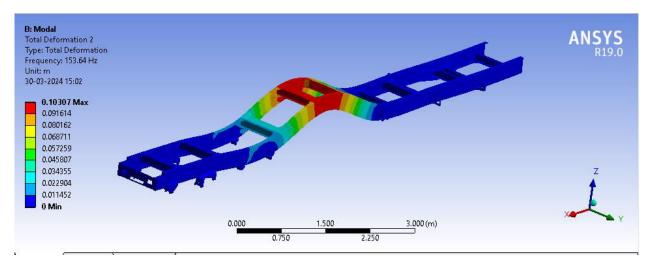


Fig 3:Mode2 natural frequency 153.64Hz

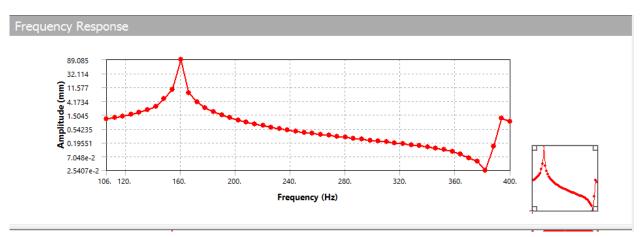


Fig 4: harmonic analysis of chassis



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#### V. SINGLE TUNED MASS DAMPER

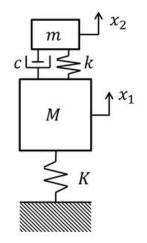


Fig 4: single tuned mass damper

A single tuned mass damper reduces mechanical vibrations in systems to prevent damage. It matches its natural frequency to the excitation frequency, stabilizing displacements caused by harmonic vibrations.

#### VI. TUNED MASS DAMPER DESIGN

Mass of truck Chassis = 0.85 tones. Adding mass for tune mass damper is 3 % of total mass of chassis. Adding mass=0.03 X 0.85=25.5 Kg First natural frequency of chassis is noted as 160 Hz, according this sfiffness is calculated using below.  $f_n = \frac{1}{2 \pi} \sqrt{\frac{K}{Mt}}$ 

Stiffness K is calculated as 25715.099 N/m

#### VII. DESIGN MODIFICATION OF CHASSIS

The tuned mass damper, combining spring and damper systems, reduces mechanical vibrations. Identified through modal analysis at 160Hz, high chassis amplitude necessitates its use. Widely applied in mechanical and civil engineering, TMDs consist of mass, spring, and dashpot components for vibration control. They vary in size and configurations like pendulum or liquid dampers are also used for mitigation.

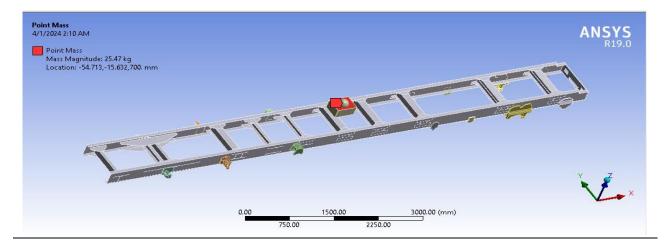


Fig 5: Design modification of chassis



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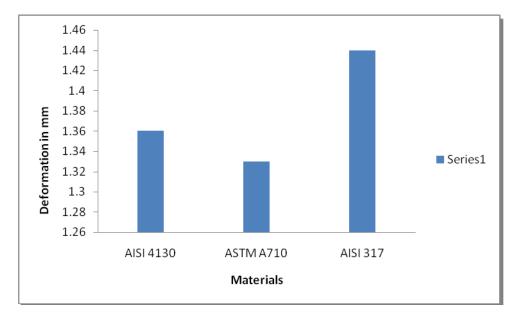
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#### VIII. FINAL RESULTS

|             |      | AISI   | ASTM    |          |
|-------------|------|--------|---------|----------|
|             |      | 4130   | A710    | AISI 317 |
| deformation | mm   | 1.36   | 1.33    | 1.44     |
| stress      | M.Pa | 225.06 | 225.06  | 227.3    |
| Strain      |      | 0.0016 | 0.00164 | 0.0018   |

Table-2 results of the static analysis



#### **GRAPH** :Deformation for different Materials

|   | Mode | Frequency [Hz] |
|---|------|----------------|
| 1 | 1.   | 145.42         |
| 2 | 2.   | 153.64         |
| 3 | 3.   | 206.38         |
| 4 | 4.   | 218.83         |
| 5 | 5.   | 225.95         |
| 6 | 6.   | 353.87         |

Table-3 results of the modal analysis

#### **IX. CONCLUSION**

In this work numerical analysis is conducted on the vehicle framework. Catia v5 version software program is accustomed to create the particular model and it also was brought in into analysis software ANSYS. static, Modal, Harmonic test provides performed on the chassis. Within order to conduct anxiety and vibration analysis in the existing truck chassis, finite element method is used to assess the static in addition to dynamic structural behaviours of truck chassis. The stress/strain distribution and natural frequencies were calculated over the chassis. Maximum tension and

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strain levels are usually found in front section regarding chassis, where engine and transmission are installed. in the structural analysis, the chassis has minimum deformation and stresses for A710 materials. And it was selected for optimum materials, based on this modal. Harmonic and tune mass damper analysis are performed. Tune mass damper is helped to reduce the vibration amplitude.

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