

An ISO 3297: 2007 Certified Organization

Volume 4, Special Issue 3, March 2015

First National Conference on Emerging Trends in Automotive Technology (ETAT-2015)

Organized by

SAE INDIA Velammal Collegiate Club, Velammal Engineering College, Chennai, India on 20th March 2015

Conversion of Naturally Aspirated Genset Engine to Meet III A Norms for Tractor Application by Using Turbocharger

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ABSTRACT: The small power diesel engines are most demanding product in Indian market for power genset and Tractor applications. The basic requirements of the end user in off-road market are good response, transient performance, better low speed torque, better fuel efficiency and smooth operation of the engine. Additional requirements needed to sustain the market are downsizing of the engine, higher power to weight ratio and increased life of the engine.

Turbo charging applied to off road diesel engines is a promising solution for enhancing rated power, low end torque, transient performance, better low-speed torque, optimized fuel efficiency and downsizing diesel engines. A trade-off is required to match some conflicting issues, i.e. overall dimensions, cost, emissions control and performance in order to sustain. Future Diesel engine emission standards will further restrict vehicle emissions, particularly nitrogen oxides.

In the present work, reference is made to 1.29L, 2 cylinder in-line diesel engine for genset application and its conversion to off-road application satisfying all the needs of the market is attempted by the application of turbo charging. The development methodology adopted to achieve high pressure ratio turbo charging is discussed along with operating limitations. Turbo charging the internal combustion engine has become a key point in both reduction of pollutant emissions and improvement in engine performance. The matching between turbocharger and engine is difficult in two cylinder engines because of the highly unsteady flow and off-design conditions the turbocharger works with.

An elaborate study is done on this 2 cylinder compact diesel engine for the optimization of performance with changes in nozzle geometry, static injection timing, change in injector end pressure, nozzle tip protrusion and exhaust gas recirculation to achieve TREM IIIA norms in naturally aspired engine. The emission values are documented in the present work and further the behavior of the engine and the ease of meeting emission norms by turbo charging and intercooling is also done to clearly indicate the need of supercharging in off-highway applications

KEYWORDS: Good response, Transient performance, Turbo charging, compact diesel engine, Trem III A.

I. INTRODUCTION

A diesel engine (also known as a compression-ignition engine) is an internal combustion engine that uses the compression to initiate ignition and burn the fuel that has been injected into the combustion chamber. This contrasts with spark-ignition engines such as a petrol engine (gasoline engine) or gas engine (using a gaseous fuel as opposed to gasoline), which use a sparkplug to ignite an air-fuel mixture. The diesel engine has the highest thermal efficiency of any standard internal or external combustion engine due to its very high compression. Low-speed diesel engines (as used in ships and other applications where overall engine weight is relatively unimportant) can have a thermal efficiency that exceeds 50%.



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II. TURBO CHARGING

Engine power is proportional to the amount of air and fuel that can get into the cylinders. All things being equal, larger engines flow more air and will produce more power. If a small engine has to perform like a big engine, or make the bigger engine produce more power, ultimately more air is to be drawn into the cylinder.

III. ENGINE DETAILS

The engine used is a 2 cylinder constant speed direct injection diesel engine capable of generating 11.8 kW. The technical specification of the base engine is shown below

No of Cylinder	2
	11.8 kW @ 1500 rpm (16 bhp
Power Rating	@1500 rpm)
Engine Speed	1500
Bore and Stroke	95 mm * 91 mm
Cubic Capacity	1.3 ltrs
Cooling System	Water cooled
Aspiration type	Naturally aspired
Compression Ratio	18.3:1
Fuel Injection pump	Bosch , Inline
Emission Norms	CPCB- I

IV. EXPERIMENTAL SETUP

The experimental setup consists of engine, dynamometer, gravimetric fuel measurement system, rectifier (converts AC to DC according which is used to start the engine), water cooling system, inlet depression and exhaust back pressure control valve. Airflow meter, blow by meter, testo emission meter, smoke Meter and crankcase pressure measuring instrument are the other instruments used for testing. The various parameters such as speed, torque, fuel time, fuel temperature, lube oil temperature, lube oil pressure, engine coolant temperature, inlet air temperature, exhaust gas temperature are indicated in the control panel.

DYNAMOMETER

The engine is coupled to an Eddy current type dynamometer. It is connected to Engine by means of Cordon Shaft. The specification of the dynamometer is shown below

Make	Benz system (Dynalec Controls)					
Туре	Eddy current					
Model	ECB 300					
Capacity	200 Nm					
Dynamometer Constant	9549.3					
ВНР	(Torque x Speed)/Dyno constant					



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PRESSURE MEASUREMENT

U-tube glass manometer is used to measure pressure at required locations and the range required is shown below

Parameter	Sensor	Range	Fluid used			
Inlet depression	U-tube manometer	0-300 mm	Water column			
Air after compressor	U-tube manometer	0-800 mm	Mercury column			
Air after Intercooler	U-tube manometer	0-800 mm	Mercury column			
Exhaust back pressure	U-tube manometer	0-300 mm	Mercury column			
Lube oil pressure	Transducer	0-10 bar	Strain gauge			

TEMPERATURE MEASUREMENT

Thermocouple is used to measure temperature at required points. Digital indicators are available at the control panel.

Sensor	Туре	Range				
Thermocouple	K type	-200 to 1000 c				

EMISSION MEASUREMENT

A portable emission meter is used to determine the exhaust emissions from the engine. Testo 350XL is the instrument used and is capable of measuring Carbon monoxide, Nitrogen oxides, Unburnt hydrocarbons in parts per million and it also measures oxygen and carbon di oxide content in percentage of exhaust flow.

MEASUREMENT OF AIR FLOW

An air flow meter is used to determine the mass of air that the engine takes. The specification of the air flow meter is shown below

Make	Range	Sensitivity	Accuracy		
Sensyflow	0-620 kg/hr	0.1 kg/hr	5 % FSR		

TEST BED SETUP

The experimental setup consists of The test engine Dynamometer Diesel tank Air filter Gravimetric fuel flow meter Sensyflow air flow meter Testo 350xl-flue gas analyser Temperature and pressure sensorsControl panel.



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V. EXPERIMENTS ON TURBOCHARGED ENGINE

A turbocharger was fitted in the engine and the performance with emission data were recorded to study the effect of turbo charging on engine exhaust emissions. In addition to the above, an attempt was also made to analyze the effect of varying combustion affecting parameters such as nozzle geometry, static injection timing, nozzle tip protrusion, and high pressure pipe change on the test engine. Table 4.3 shows the list of trials done to optimize the torque and power trend of the engine. Hence, these experiments are grouped together and are termed as torque shaping experiments.

Torque trend shaping experiments

SI NO	INJECTOR			SIT	NTP	HIGH PRESS PIPE	SURE		PURPOSE OF TRIAL			
	Hole size	Cone angle	No. of spray			ID	OD	L				
	mm	deg		deg	mm	mm	mm	mm				
1	0.19	142	5	13	2.5	1.8	6	260	As it is –Torque back up 30% will result in high free acceleration smoke.			
2	0.21	142	5	13	2.5	1.8	6	260	Injector trial- Increased hydraulic through flow to reduce torque back up.(torque curve shaping)			
3	0.21	142	5	13	2.5	2	6	260	HPP trial- Reduce residual pressure for reduced torque backup(torque cure shaping)			
4	0.197	150	6	13	2.5	2	6	260	Injector trial- Increased hydraulic through flow to reduce torque back up.(torque curve shaping)			
5	0.197	150	6	13	1	2	6	260	NTP trials – bowl geometry matching to reduce smoke			
6	0.17	144	5	10	2.5	1.8	6	260	Injector trial			



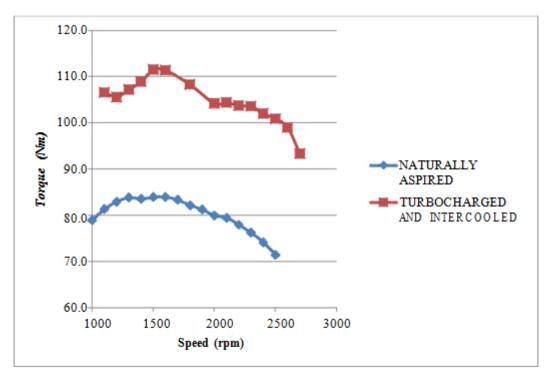
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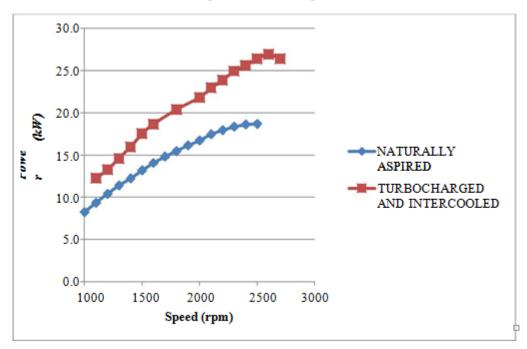
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Comparison of Torque



Comparison of Power



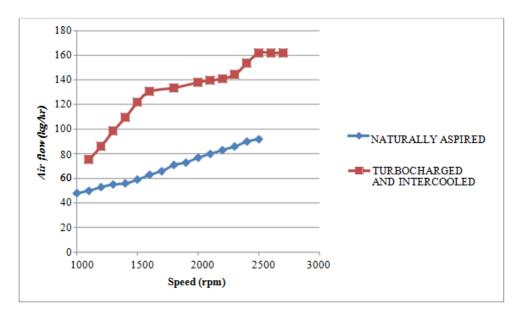
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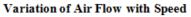
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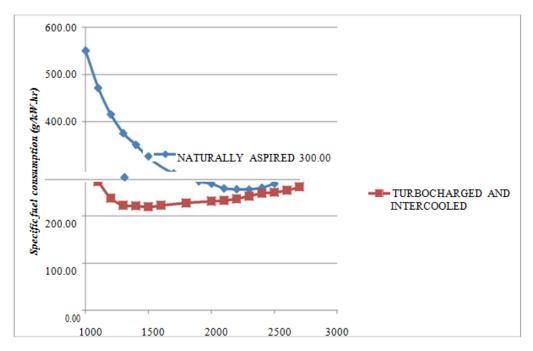
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Variation of specific fuel consumption with Speed



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	ENG	PUMP			NOZZLE				RESULTS					
				Plunge r	Plunge r			cone	No. of	Seating				
Trial	Rating	Cam_shaft	SIT	diame	ter lift	Element	hole size	angle	sprays	was he r	Nox	Со	Smoke	SFC
			deg BTD	C mm	mm		mm	deg		mm	g/kw.hr	g/kw.ł	r FSN	g/kw.hr
1	18.4kW@2500 rpm	regular	15	8	9	LLA	0.17	144	5	2.5	9210	6.404	32	249.52
2	18.4kW@2500 rpm	regular	11	8	9	ША	0.21	142	5	25	7.090	2,730	41	273.11
3	18.4kW@2500 rpm	regular	13	8	9	LLA	0.21	142	5	25	8.070	3.120	3.6	260.00
4	18.4kW@2500 rpm	regular	15	8.5	9	PE	0.21	142	5	25	10.410	3.190	3.1	249.38
5	18.4kW@2500 rpm	regular	15	8.5	9	PE	0.17	144	5	25	12.330	2.404	1.59	235.02
6	18.4kW@2500 rpm	regular	12	8.5	9	PE	0.17	144	5	25	10,740	2800	2747	240.00
7	18.4kW@2500 rpm	regular	12	8.5	9	PE	0.19	142	5	25	9.961	3,761	1.82	241.00
8	18.4kW@2500 rpm	TINER (lift 0.7mm)	12	8.5	9	PE	0.17	144	5	25	8.48	18.11	85	296
9	25.7kW@2500rpm	regular	17	8.5	9	PE	0.197	150	6	1	11.400	1.100	12	237.00
10	25.7kW@2500rpm	regular	15	85	9	PE	0.197	150	6	1	10.310	1200	1.4	239.00
11	25.7kW@2500rpm	regular	12	85	9	PE	0.197	150	6	1	7.830	1.500	1.83	249.00
12	25.7kW@2500rpm	regular	8	8.5	9	PE	0.197	150	6	1	6.44	2.54	2.36	240.00

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