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Attitude Control using a Disturbance Observer Based Controller

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ABSTRACT: In this paper a Reusable Launch Vehicle in atmospheric phase is considered. A disturbance observer based PD controller is proposed for controlling the attitude of the system considered. This method is an effective way of achieving disturbance rejection and robustness to plant uncertainties. The disturbance observer based control is used for observing the flexibility dynamics and stabilizing the flexible mode RLV. The disturbance observer makes the actual flexible plant dynamics similar to that of nominal rigid body within a limited frequency range. The classical gain stabilization approach for stabilizing bending mode is also presented. Both methods are compared and advantages of disturbance observer based PD control over the gain stabilized PD control are discussed.

KEYWORDS: Launch Vehicle, flexibility, DOBC, LQR.

I. INTRODUCTION

Reusable launch vehicle (RLV) is capable of launching a vehicle into space more than once. The technical challenges of designing a system to fly to orbit and return are monumental. The dynamic of the vehicle is highly interactive and the vehicle is influenced by a lot of uncertainties. It is also affected by unknown external disturbances. Problems associated other than complex dynamics include path constraints, heat rates, structural loads, and dynamic pressure. Main challenges include the uncertainty and external disturbance induced from the complex atmosphere characteristics. Controller designed for this kind of plants should have rapid adaptability and sufficient robustness.

Gain scheduling was widely employed in flight control systems [1]. The method is not always effective when applied to RLV. Because RLV may need aggressive maneuvering in order to dissipate excessive energy. Also the operation of RLV can be far away from the equilibrium point. Then Dynamic inversion method was used to overcome the disadvantages of Gain Scheduling. Non-linear dynamic inversion (NDI) is a method were we invert the non-linearities and linearize the non-linear system [4]. The robustness properties of the dynamic inversion are questionable especially when considering the uncertainties involved. Incremental dynamic inversion method was able to provide better robustness compared to NDI [3]. The basic idea of the methods is to cancel the nonlinear dynamics via mathematical transformation.

Recently, a high-order SMC technique has been employed in reentry attitude control. Sliding mode disturbance observer was used. Also, it provides a good robustness to external disturbances, model uncertainties, and different mission trajectories. But the drawback is that attitude tracking is achieved within an infinite time rather than finite time. Slow convergence rate and poor disturbance rejection performance. But, when NDI is used along with sliding mode control, advantages are increased [5]. Limited time convergence, robustness, easy implementation are some main advantages.

The idea of disturbance observer was put forward by K.Ohnishi in 1987. Once the disturbances are estimated, control of dynamic system with disturbances becomes easier. Disturbance Observer Based Control (DOBC) has found applications in various fields such as servo motor control, robotics, flight control, magnetic levitation etc. The disturbance observer (DOB) treats the difference between the actual plant and nominal model which occurs due to parameter variations and/or external disturbances as an equivalent lumped disturbance [9]. According to the disturbances observed by DOB, the same amount of compensation to control input is introduced to realize strong



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robustness. Robust autopilot design for bank-to-turn missiles under disturbances and uncertainties is investigated in [11] using disturbance observer concept. A non-linear disturbance observer based robust flight control for an air-breathing hypersonic vehicle is presented in [12] which is capable of eliminating mismatched disturbances also. The application of disturbance observer to a flexible spacecraft subjected to uncertainties and external disturbances is given in [13].

The organization of the paper is as follows: The second section gives a small description of the plant. In the third section, LQR controller response for the plant is discussed. In the fourth section, the design of PD controller is shown. In the fifth section a notch filter is designed to attenuate the flexibility dynamics. The control signal entering the plant is analyzed in the sixth section. The response of the plant for a PD-Notch filter is analyzed in the section. Finally the ninth section includes the conclusion.

II. SYSTEM MODEL

The launch vehicles are broadly divided into two main types: Re-Usable Launch Vehicle and Extinguishable Launch vehicle. The general equations describing motion are initially obtained by ordinary differential equations [16]. These are equations of force and momentum incorporating flexibility and sloshing. These equations are valid only for short time periods-on the order of a few seconds. Vehicle properties such as mass, centre of gravity location, and moment of inertia are assumed constant in this interval. But while applying this assumptions, further complications are introduced because of the fact that the vehicle mass varies with time. The derivation of the equations of motion of a space launch vehicle-including the effects of elasticity, fuel sloshing and engine inertia maybe applied by Newton's laws directly to the entire vehicle. The general equations of force and momentum are derived and by removing the sloshing, inertia and flexible terms, the equation of rigid body is obtained. Here, the longitudinal dynamics of an RLV in atmospheric phase is considered. The equations of motion of RLV are described by continuous non-linear differential equations. Some assumptions are taken to obtain a linearised model of a launch vehicle. The time varying mass and inertial properties are assumed to be invariant for a short period of time. The deviations from the reference trajectory are assumed to be small. The attitude dynamics are assumed to be de-coupled. The non-linearities of sensors and actuators are neglected.

The rigid body dynamics is represented by a third order system where the states are given by the attitude (θ) , rate of

attitude (θ), and angle of attack (α). But when the vehicle is flexible, two additional states (q and \dot{q}) are added for each bending mode. This flexibility problem occurs when the sensing instrumentation (gyros) picks up not only the rigid body motion but also the local elastic distortions. Theoretically, because the vehicle is continuous body, the elastic motion is described by a system with infinite degrees of freedom. In practice, either a truncation of the infinite series or a lumped mass model is used to yield the system with a finite number of elastic modes. The number of modes that are significant in a given situation depends on the bandwidth of the primary dynamic modes. Here, only one bending mode is considered. Hence there are a total of 5 states and two outputs (outputs from position gyro and rate gyro). The state equation and output equation of a flexible launch vehicle is given of the form:

 $\dot{x}(t) = Ax(t) + Bu(t)$

$$y(t) = Cx(t) + Du(t)$$

Due to the flexible and lightweight structure of RLV, distortions of sensor measurements and stability problems may arise. Hence, proper stabilization should be done. It is assumed that both attitude and attitude rate signals are available for feedback.

Due to the presence of bending mode dynamics due to flexibility of the launch vehicle, it has been found that there are unwanted dynamics while taking the open loop response of the plant.



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Fig 1.Open loop bode response of attitude.

It has been found that there is an unwanted flexible dynamics at 39.7 rad/s as shown in figure 1. The system is unstable with an infinite gain margin and phase margin at 3.44 rad/s is -2.65.

III. LQR CONTROLLER

Linear Quadratic Optimal Control Theory (LQR) is the most important and most comprehensive of a class of optimization based synthesis problem to obtain the performance index function by not only taking into account the performance requirements but also energy requirements. Quadratic performance measures have been extensively used in the design of optimal control systems because of its mathematical tractability and the ability to accommodate realistic functions such as energy functions and squared errors or deviations. When squared errors are used as performance measures, larger errors are given a much higher penalty than smaller errors as the squares of large quantities are very large and squares of small quantities are very small.

Consider a plant having a linear state equation. In the optimal control problem, it is desirable to determine a control signal that will minimize the performance measure. The optimal control is derived as: u = -Kx(t) where K is the

optimal gain matrix. The performance measure to be minimized is taken as $J = \int_{0}^{\infty} [x^{T}Qx + u^{T}Ru]dt$. Here Q is

positive semi definite while R is positive definite.

Finally after mathematical analysis, the linear optimal feedback gain matrix is obtained as $K = R^{-1}B^T P$, where P(t) is obtained after solving the equation $A^T P + PA - PBR^{-1}B^T P + Q = 0$.

In simulation, in both the cases R is taken as 1. Q is taken as gain multiplied by the transpose of C matrix with C matrix. The gain is taken as 1 in flexible plant simulation while the gain is taken as 2 in rigid body simulation. In both the cases the output is found to give stable response with a settling time of 4.41s for attitude in rigid body dynamics and a settling time of 4.95s for attitude in flexible RLV.

But the problem with LQR is that it cannot be used in all cases, it can be used only under limited conditions. LQR does not give satisfactory response in the case of uncertainties and also it has a large steady state error.



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Fig 2.LQR response for rigid body dynamics of RLV



IV. PD CONTROLLER

The two outputs from the launch vehicle are the attitude and the rate of attitude. It is assumed that the attitude and rate of attitude are available for feedback. The proportional and derivative constants are designed via pole placement. The desired damping coefficient and natural frequency are taken as 1 and 6rad/s respectively. The designed values of proportional and derivative gains are obtained as 2.3391and 0.2869 respectively. These gains ensure sufficient stability margins. The closed loop system with the plant and controller is shown in figure 4. The step response of the closed loop system with the plant as the rigid body dynamics is obtained and shown in the figures 5, 6. The step response of attitude has a settling time of 5.18s, rise time of 0.361s and a steady state error of .5%. But when the plant is a flexible RLV, the result is an unstable system.



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Fig 4. Closed loop system with plant and rigid body dynamics



Fig 5.Step response of PD controlled nominal plant(heta)

Fig 6.Step response of PD controlled nominal plant(heta).

V. FLEXIBILITY DYNAMICS STABILIZATION USING NOTCH FILTER

The frequency response of the rigid body including the flexibility dynamics is shown in the figure. The presence of flexibility dynamics degrades the controller performance and even drives the system to instability.



Fig 7.Open Loop Frequency response of attitude and rate of attitude

The use of a PD controller alone cannot stabilize the flexible RLV dynamics. When the flexible RLV plant is simulated, it is found that there is a bending mode at frequency 39.7rad/s. In order to stabilize the plant, a notch filter is designed at this frequency. A notch filter is used for attenuating the signal at one particular frequency. The transfer function of a typical notch filter is given as follows:

$$T.F = \frac{s^{2} + 2\zeta_{1}\omega_{n}s + \omega_{n}^{2}}{s^{2} + 2\zeta_{2}\omega_{n}s + \omega_{n}^{2}}$$

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Here ω_n is the natural frequency, ζ_1 and ζ_2 are the damping values. $\frac{\zeta_1}{\zeta_2}$ denotes the depth of the notch.



Fig 8.Bode plot of PD controlled flexible launch vehicle

This is the method of bending mode stabilization using gain stabilization. Usually gain stabilization is used for stabilizing higher order bending modes. A notch filter is designed as follows:

By taking
$$\omega_n = 39.83$$
,
 $\zeta_1 = 0.01$, $F(s) = \frac{s^2 + .7966s + 1586}{s^2 + 79.66s + 1586}$.
 $\zeta_2 = 1$

The frequency plot of notch filter is given by figure 9.



VI. CONTROL SIGNAL WITH AND WITHOUT NOTCH FILTER

Before the output of the plant is analyzed, the control signal is analyzed. The plant is analyzed in an open-loop fashion for observing the input to the plant. The control signal is found to have unwanted dynamics as shown in figure 10. The output of the notch filter is found to attenuate these unwanted dynamics at that frequency as shown in figure 11.



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Fig 10.Control signal with PD controller

Fig 11.Control signal with PD-Notch filter

VII. PD CONTROLLER WITH NOTCH FILTER

The gain stabilized PD controller is capable of stabilizing the flexible RLV dynamics. The frequency response shows that very good stability margins are achieved. The step response shows that it is similar to rigid body dynamics.



Fig 12.Step response of PD-Notch controlled plant (heta)



Fig 13.Step response of PD-Notch controlled plant (heta)



Fig 14.Bode plot of attitude with PD-Notch filter

It is found that the output bode is having a smooth response with less dynamics and was found to be closed loop stable with phase margin of 132rad of and gain margin of 11.5dB. The step response of attitude is found to have a rise time of 0.274s and a settling time of 5.15s with a steady state error of 0.5%.



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Dept. of EEE, College of Engineering, Trikaripur (CAPE, Government of Kerala), Cheemeni, Kasaragod, Kerala-671313, India VIII. DISTURBANCE OBSERVER BASED PD CONTROL

Disturbance Observer:

Disturbance observer is used to estimate the disturbance. This disturbance is fed back and added along with the outputs from other parts of whole system. The disturbance observer can estimate not only the external disturbance but also the internal disturbances caused by uncertainties. Consider a SISO linear system depicted by the following frequency domain form:

$$Y(s) = G_P(s)[U(s) + D(s)]$$

Where Y(s) is the controlled output, U(s) is the control input; D(s) is the disturbance and $G_p(s)$ is the model of the plant.

The block diagram of a disturbance observer in frequency domain is given as shown in figure 15.



In the figure $G_n(s)$ represents the nominal plant and Q(s) represents the filter of disturbance observer. Here the selection of Q(s) is an important thing. It is usually taken as a low pass filter. Also Q(s) is taken in such a way that it's steady state gain is one and the relative degree is greater than $G_n^{-1}(s)$. These conditions should be ensured before implementing the disturbance observer. The mathematical proof can be shown below:

The output can be represented as $V(s) = G_{-}(s)U_{-}(s) + G_{-}(s)D(s)$

$$T(s) = G_{uy}(s) O_{c}(s) + G_{dy}(s) D(s)$$

Where $G_{uy}(s) = \frac{G_{P}(s)G_{n}(s)}{G_{n}(s) + Q(s)[G_{P}(s) - G_{n}(s)]}$
 $G_{dy}(s) = \frac{G_{P}(s)G_{n}(s)[1 - Q(s)]}{G_{n}(s) + Q(s)[G_{P}(s) - G_{n}(s)]}$

When the filter is selected as a low pass filter, when frequency tends to zero Q(s) tends to one, hence we get

 $\lim_{\omega \to 0} G_{uy}(j\omega) = G_n(j\omega)$ $\lim_{\omega \to 0} G_{dy}(j\omega) = 0$

Hence, it is understood that, the system with disturbance observer behaves as a nominal plant at low frequency region. The performance of disturbance estimation is mainly determined by the design of low pass filter Q(s).



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Fig 16.General Disturbance Observer or Estimation Filter

The equivalent structure of a general disturbance observer is shown in the figure 16. The control signal to the plant is found to have the inverted disturbance as shown in figure 17. This when added to the original disturbance gets cancelled off and hence the stability of the plant is assured.

The control signal plots and the output plots of the general disturbance observer are as shown in figures 17 and 18.



Fig 17.Control Signal

Fig 18.Output

The control signal is having the exact inverted flexible dynamics at 39.7 rad/s. This when given to plant, will result in a stabilized response.

Disturbance Observer Based PD Control:



Fig 19.Disturbance Observer based PD control



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Once the disturbance observer based PD controller was implemented, it was found that the system was stabilized. The step response of attitude was found to have a settling time of 5.25s, rise time of 0.381s and a steady state error of 0.5% as shown in fig.20.



The frequency response was found to have a closed loop stable response having specifications with a gain margin of 13.3dB and a phase margin of 135rad.



If the bending mode parameters are not certain, the notch filter fails to attenuate it completely whereas a disturbance observer is capable of estimating and stabilizing the flexible plant even in the presence of uncertainties. Not only bending mode uncertainties but the effect of any other parameter changes can also be compensated by using the disturbance observer since it makes the actual plant behave like a nominal one by cancelling the mismatch between the two.

IX. CONCLUSION

Here a disturbance based observer based controller is designed for a flexible RLV in atmospheric phase. Though the performance of DOBC is similar to that of classical gain stabilized controller, in case of uncertainty in bending mode parameters, the gain stabilized controller needs to be re-designed but a disturbance observer performs well under parameter variations. Hence, it is a better method compared to the classical gain stabilization technique.

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