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Design and Analysis for Developing an Efficient Centrifugal Clutch Used In Automobile to Minimize the Deteriorating Effects

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ABSTRACT: Clutch is the most important component of any automotive machine. It is a mechanism which is used to transmitting the rotation from one shaft to another shaft by engagement and disengagement of the same. It is a connection between engine and transmission system which transmit power in the form of torque from engine to the gear assembly. When vehicle is in motion clutch is first disengaged for the drive to allow for gear selection and then again engaged smoothly to power the vehicle to transfer torque to the driving wheels.

In the present work SolidWorks software is used for modelling of two different types centrifugal clutch and ANSYS 14 workbench for static analysis then comparison is made for the maximum principle stress developed and maximum deflection in the three shoe clutch and rigid body type clutch which can be used in automotive vehicles.

KEYWORDS: Centrifugal Clutch, FEM, Solid works, Ansys workbench.

I. INTRODUCTION

Clutch

A clutch is a machine member used to connect a driving shaft to a driven shaft so that the driven shaft may be started or stopped without stopping the driving shaft. It is generally seen in automobiles. A little consideration will show that in order to change gears or to stop the vehicle, the driven shaft should not be connected with the energy source, but at the same time the engine should continue to run so it becomes necessary that the driven shaft should be disengaged from the driving

shaft. The engagement and disengagement of the shafts is obtained by means of a clutch which is operated by a lever.

Types of Clutches

Following are the two main types of clutches commonly used in engineering practice :

- 1. Positive clutches
- 2. Friction clutches

Positive Clutches

The positive clutches are used for positive drives. Jaw or claw clutch is the simplest type of positive clutch. The jaw clutch permits one shaft to drive another through a direct contact of interlocking jaws. It is having two halves, one of which is permanently fastened by a sunk key to the driving shaft. The other half of the clutch is movable and free to slide axially on the driven shaft. The jaws of the clutch may be of square type as shown in Fig. 1(a) or of spiral type as shown in Fig. 1(b). A square jaw type is used where engagement and disengagement in motion and under load is not necessary. This type of clutch will transmit power in either direction of rotation. The spiral jaws may be left-hand or right-hand, because power transmitted by them is in one direction only. This type of clutch is occasionally used where the clutch must be engaged and disengaged while in motion. The use of jaw clutches are frequently applied to sprocket wheels, gears and pulleys. In such a case, the non-sliding part is made integral with the hub.



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Fig.1 Jaw clutches

Friction Clutches

A friction clutch has its application in the transmission of power of shafts and machine parts which must be started and stopped frequently. It is likewise utilized when force is to be conveyed to machines which is partially or fully loaded. The frictional force is utilized to begin the driven shaft from rest and gradually brings it up to the proper speed without excessive slipping of the friction surfaces. In automobiles, friction clutch is utilized to interface the engine to the drive shaft. In operation consideration ought to be taken so that the friction surfaces engage easily and gradually bring the driven shaft up to appropriate speed. The proper alignment of the bearing must be maintained and it should be located as close to the clutch as possible. It likewise be noticed that:

1. The contact surfaces should develop a frictional force that may pick up and hold the load with reasonably low pressure between the contact surfaces.

2. The heat of friction should be rapidly dissipated and tendency to grab should be at a minimum.

3. The surfaces should be backed by a material stiff enough to ensure a reasonably uniform distribution of pressure.

Considerations in Designing a Friction Clutch

The following considerations must be kept in mind while designing a friction clutch.

1. The suitable material forming the contact surfaces required to be chosen.

2. The moving parts of the clutch should have low weight in order to minimise the inertia load, especially in high speed service.

3. The clutch should not require any external force to maintain contact of the friction surfaces.

4. The provision for taking up wear of the contact surfaces must be provided.

5. The clutch should have provision for facilitating repairs.

6. The clutch should have provision for carrying away the heat generated at the contact surfaces.

7. The projecting parts of the clutch should be covered by guard.

Types of Friction Clutches

Though there are many types of friction clutches, yet the following are important from the subject point of view :

- 1. Disc or plate clutches (single disc or multiple disc clutch),
- 2. Cone clutches, and
- 3. Centrifugal clutches.

Centrifugal Clutch

The centrifugal clutches are usually incorporated into the motor pulleys. It consists of a number of shoes on the inside of a rim of the pulley, as shown in Fig. 2. The outer surfaces of the shoes are covered with a friction material. These shoes, which can move radially in guides, are held against the boss (or spider) on the driving shaft by means of springs. The springs exert a radially inward force which is assumed constant. The weight of the shoe, when revolving causes it to exert a radially outward force (i.e. centrifugal force). The magnitude of this centrifugal force depends upon the speed at which the shoe is revolving. A little consideration will show that when the centrifugal force is less than the spring force, the shoe remains in the same position as when the driving shaft was stationary, but when the centrifugal force is equal to the spring force, the shoe is just floating. When the centrifugal force exceeds the spring force, the shoe moves outward and comes into contact with the driven member and presses against it. The force with which the shoe presses



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against the driven member is the difference of the centrifugal force and the spring force. The increase of speed causes the shoe to press harder and enables more torque to be transmitted.



II. OBJECTIVE OF WORK

Following are the major objectives of the given work

- i) Firstly, to facilitate an automatic transmission system, which must has high strength and low deformation as compared to existing system.
- ii) Secondly, to introduce light-weight vehicle transmission system where the transmission belt and pulleys are substituted with a centrifugal clutch and planetary gear assembly which is more compact and which avoid loss of power when operating at high speed.
- iii) Lastly, to provide an automatic transmission which is light in weight, compact and economical of manufacturing.

III. MODELLING OF CENTRIFUGAL CLUTCH

In this section the modelling of Centrifugal clutch is done by SolidWorks modelling software. SolidWorks enables to design models as solids in progressive 3D solids modelling environment. Solid models are geometric models that offer mass properties such as volume, surface area and inertia.

Figure 3 shows model of rigid body type clutch. It have only one rotating component whereas figure 4 elaborate model of Three shoe clutch which containing three rotating components.



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Fig.3 Model of Rigid body type



Fig.4 Model of Three shoe clutch

IV. ANALYSIS OF CENRIFUGAL CLUTCH

The ANSYS software is a handy tool that has the capability to decide the type of elements that are required for the analysis purpose. The Finite Element Method is a mathematical tool used to compute ordinary equations and partial differential equations. Since it is a numerical tool, it has the ability to solve the complex problems that are represented in differential equations form. Mesh generation is one of the most critical aspects of engineering simulation. Too many cells may result in long solver runs, and too few may lead to inaccurate results. ANSYS Meshing technology provides a means to balance these requirements and obtain the right mesh for each simulation in the most automated way possible. ANSYS

Meshing technology has been built on the strengths of stand-alone, class-leading meshing tools. The strongest aspects of these separate tools have been brought together in a single environment to produce some of the most powerful meshing available. This helps in automatic choosing of elements and nodes and thus simplifies the job of FEM model creation. The meshing is performed by opening the ANSYS meshing application in workbench environment wherein the sizes of the elements are controlled.

A 3D model of a two different design of centrifugal clutch are use for analysis in ANSYS 14 workbench. The loading conditions are assumed to be static. The element selected is 45 tetrahedral. Finite element analysis is carried out on rigid body type clutch and three shoe clutch. The material properties for were taken from the reference papers. From the analysis the Maximum Principle stress and maximum deformation were determined.



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Figure 5 and Figure 6 shows three shoe clutch and rigid body type clutch respectively imported in ansys for static analysis.



Fig. 5 Three shoe clutch model imported in Ansys for analysis



Fig. 6 Rigid Body type clutch model imported in Ansys for analysis



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V. RESULTS AND DISCUSSION

a) Maximum Principle stress



Fig. 7 Maximum Principle stress of three shoe clutch



Fig. 8 Maximum Principle stress of Rigid body type clutch

Fig 7 and fig. 8 shows maximum principle stress as 45.122 MPa and 293.132 MPa for three shoe cutch and rigid body type clutch respectively.



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b) Total Deformation



Fig. 9 Total deformation of Three shoe clutch



Fig. 10 Total deformation of Rigid body type clutch

Fig. 9 and fig. 10 shows maximum Deformation as 0.33029 mm and 5.9583 mm for three shoe cutch and rigid body type clutch respectively.

VI. CONCLUSION

It can be concluded from the result that the stress generated and deformation in three shoes clutch is less as compared to rigid type clutch system.

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Туре	Three shoes clutch	Rigid body type clutch
Max principle stress generated	45.122 MPa	293.32 MPa
Maximum Deformation	0.33029 mm	5.9583 mm
Location of Maximum deformation	Clutch drum	Bearing support



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Analysis also expressed the location of maximum deformation which is at clutch drum for three shoe clutch and bearing support for rigid body type clutch.

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